# ESTABLISHING THE MOTION FUNCTIONS FOR AN ASSISTIVE ROBOT FOR PEOPLE WITH MOTOR DISABILITIES 

Claudiu SCHONSTEIN, Claudiu-Ioan RUSAN


#### Abstract

Nowadays, robots and robotic mechanisms are also a great technological challenge and as a result they can be found in all fields of activity, such as service, management, industrial, agricultural and medical. In the medical field, in addition to robots assisting doctors in various operations, there are also robots assisting patients with various traumas and strokes to regain their mobility, so the involvement and demand for robots in the medical field is growing. The assistive robot presented in this paper has the potential to provide support for a wide range of tasks related to physical assistance, therapies and rehabilitation.


Key words: motion functions, assistive robot, assistant robot, locomotor disabilities, motor disabilities, robot therapies, medical robots.

## 1. INTRODUCTION

Regarding medical robots used in the care and rehabilitation of people with different disabilities, they can be classified into:

- robots designed for functional readjustment (disability correction) - when accidents or even illness impair the body's abilities, practically replace kinesiotherapy or to remedy organic deficiencies by means of exoskeletons (in the case of amputated limbs or tetraplegia), the robot replacing those defective or missing body parts;
- robots designed for social rehabilitation these are used to provide both physical and social support to disabled or elderly people. This category includes robots capable of supervising or replacing human help or when it is rejected for pathological reasons.

The development of medical robots stands out in our century as an exceptional field and perspective, so a lot of research centers have developed various mechanical systems for rehabilitation or assistance, thus putting the foundations of a new direction in research with positive results that have been an encouragement for the development of new medical systems and robots.[1]-[5]

The structure proposed for further study and development is a hybrid model, shown in figure 1 , consisting of a mobile platform on which a 2 T serial structure is mounted.


Fig. 1. The design of the robot intended for people with motor disabilities

## 2. ASSISTANT ROBOT DESIGN

The proposed structure to be developed, presented in figure 1, with the abbreviation ARPMD (Assistive Robot for People with motor Disabilities) is intended to be used to assist people with motor disabilities, but also to perform rehabilitation therapies based on robot movements in order to achieve the motor rehabilitation performance of the users.

As shown in figure 2, the robotic system is based on a mobile structure, marked with (1), on which is fixed the frame with two arms (2), which allow to perform a set of up-down or front-back movements, which helps the patient in movements.

Movement of the mobile structure (1) is achieved by means of four wheels, of which two wheels are driving wheels and the other two are for steering. The driving wheels are controlled by two DC motors, one for each wheel.


Fig. 2. The movements that the assistant robot performs

The serial structure (the arms that give control over the serial structure) are driven by four DC servomotors, as shown in figure 3, which can be adjusted according to the height of the operator, who can change the speed and direction of travel of the robot at any time.


Fig. 3. General dimensions of the robotic structure
Given the average human body size, $1,70 \mathrm{~m}$, the prototype has the following general dimensions: $1600 \times 820 \times 1400$. After modeling
the cad and applying the material qualities of the components, information was extracted and analyzed using the "mass properties" function in SolidWorks, which resulted in the mobile structure of the robotic system weigh 85 kg and the weight of the sub-assembly of the serial structure is 10 kg , totaling 95 kg . Also, as shown in figure 4, the relative center of mass to the system reference of the robot is at: $\mathrm{X}=110 \mathrm{~mm}$, $\mathrm{Y}=748 \mathrm{~mm}, \mathrm{Z}=476 \mathrm{~mm}$.


Fig. 4. Relative center of mass of the robot

## 3. MODELLING OF THE MOBILE STRUCTURE - KINEMATIC AND DYNAMIC

### 3.1 Equations of geometry and kinematics of the moving structure

Considering that the rehabilitation and assistance robot is a mobile platform, its general motion is subject to plane-parallel motion, as shown in figure 5. [6],[7]


Fig. 5. The scheme for plane-parallel motion
The displacement of the moving structure relative to the fixed reference system (O), can be
expressed by three independent parameters, according to expression:

The column vector of operational velocities expressing the absolute motion of the mobile robot is: [3]

$$
\dot{\overline{0}} X=\left[\begin{array}{lll}
\dot{q}_{1} & \dot{q}_{2} & \dot{q}_{3} \tag{2}
\end{array}\right]^{T}
$$

The mobility of the robot is ensured by two driving wheels $M_{r s 1}, M_{r s 2}$ having their center's at points $O_{1}, \quad O_{2}$ which execute rotational movements around the $Y$ axis of the robot system, as well as two other free (steering) wheels $M_{r f 1}, M_{r f 2}$ having their center's at points $O_{3}, O_{4}$ which in addition to the rotational movement around the $Y$ axis also exert a rotation along the $Z$ axis of the robot system, as shown in figure 6 .


Fig. 6. Independent parameters
The sliding constraint of the robotic platform along the Oy -axis is expressed by the relation:

$$
\begin{equation*}
\dot{y}_{R}=-\dot{q}_{1} \cdot \sin q_{3}+\dot{q}_{2} \cdot \cos q_{3}=0 \tag{3}
\end{equation*}
$$

For the structure to perform a linear displacement (translational motion) without sliding, then:

$$
\begin{align*}
& \dot{x}_{R}=\dot{q}_{1} \cdot \cos q_{3}+\dot{q}_{2} \cdot \sin q_{3} \\
& \quad=\frac{r}{2} \cdot\left(\dot{q}_{4}+\dot{q}_{5}\right)  \tag{4}\\
& q_{j i}\left(\tau_{i-1}\right)=q_{j i-1 ;} \\
& q_{j i}\left(\tau_{i}\right)=q_{j i} ;
\end{align*}
$$

From the expression (4), it follows that:

$$
\begin{equation*}
\dot{q}_{4}=\dot{q}_{5}+\frac{2 \cdot l}{r} \cdot \dot{q}_{3} ; \dot{q}_{5}=\dot{q}_{4}-\frac{2 \cdot l}{r} \cdot \dot{q}_{3 ;} \tag{5}
\end{equation*}
$$

Thus, substituting $q_{4}$ and $q_{5}$ from relation (5) into the equation gives the expressions below:

$$
\begin{gathered}
\dot{q}_{1} \cdot \cos \dot{q}_{3}+\dot{q}_{2} \cdot \sin q_{3}-\dot{q}_{4} \cdot r+\dot{q}_{3} \cdot l \\
=0 \\
\dot{q}_{1} \cdot \cos q_{3}+\dot{q}_{2} \cdot \sin q_{3}-\dot{q}_{5} \cdot r+\dot{q}_{3} \cdot l \\
=0
\end{gathered}
$$

The above expressions represent the conditions necessary for the driving wheels to perform pure, slip-free rotations.

The sliding of the freewheels along the axis $O_{y 3}, O_{y 4}$ is prevented by:

$$
\begin{gather*}
\left(\dot{q}_{1}-L \cdot \dot{q}_{3} \cdot \sin q_{3}+b \cdot \dot{q}_{3} \cdot \cos q_{3}\right) \\
+\left(\dot{q}_{2}+L \cdot \dot{q}_{3} \cdot \sin \left(q_{3}+q_{7}\right)+\right. \\
\left.\cdot \cos \left(q_{3}+b \cdot \dot{q}_{3}\right)=\sin q_{3}\right) \\
\left(\dot{q}_{1}-L \cdot \dot{q}_{3} \cdot \sin q_{3}+b \cdot \dot{q}_{3} \cdot \cos q_{3}\right) \\
\cdot \sin \left(q_{3}+q_{9}\right)+  \tag{7}\\
+\left(\dot{q}_{2}+L \cdot \dot{q}_{3} \cdot \cos q_{3}+b \cdot \dot{q}_{3} \cdot \sin q_{3}\right) \\
\cdot \cos \left(q_{3}+q_{9}\right)=0
\end{gather*}
$$

Non-slip rolling of freewheels is achieved if the following conditions are satisfied:

$$
\begin{gather*}
\left(\dot{q}_{1}-L \cdot \dot{q}_{3} \cdot \sin q_{3}+b \cdot \dot{q}_{3} \cdot \cos q_{3}\right) \\
\cdot \cos \left(q_{3}+q_{7}\right)+ \\
+\left(\dot{q}_{2}+L \cdot \dot{q}_{3} \cdot \cos q_{3}+b \cdot \dot{q}_{3} \cdot \sin q_{3}\right) \\
\cdot \\
=r_{r f} \cdot \dot{q}_{6} \quad \\
\text { sin }\left(q_{3}+q_{7}\right)=  \tag{8}\\
\left(\dot{q}_{1}-L \cdot \dot{q}_{3} \cdot \sin q_{3}+b \cdot \dot{q}_{3} \cdot \cos q_{3}\right) \\
\cdot \cos \left(q_{3}+q_{9}\right)+ \\
+\left(\dot{q}_{2}+L \cdot \dot{q}_{3} \cdot \cos q_{3}+b \cdot \dot{q}_{3} \cdot \sin q_{3}\right) \\
\\
=r_{r f} \cdot \dot{q}_{8} \quad \cdot \sin \left(q_{3}+q_{9}\right)=
\end{gather*}
$$

Applying a series of transformations on the expressions contained in (6), (7), (8), the differential kinematic constraints of the presented structure are expressed in the expressions (9)-(15):

$$
\begin{equation*}
-\sin q_{3} \times d q_{1}+\cos q_{3} \times d q_{2}+\stackrel{\circ}{a}_{i=3}^{9} 0 \times d q_{i}=0 \tag{9}
\end{equation*}
$$

$$
\begin{aligned}
& \cos q_{3} \times d q_{1}+\sin q_{3} \times d q_{2}+l \times d q_{3}-r_{r s} \times d q_{4}+ \\
& +\stackrel{9}{\mathbf{a}} 0 \times d q_{i}=0 \\
& i=5 \\
& \cos q_{3} \times d q_{1}+\sin q_{3} \times d q_{2}-l \times d q_{3}+0 \times d q_{4}- \\
& -r_{r s} \times d q_{5}+\stackrel{\stackrel{\circ}{\mathrm{a}}}{i=6} 0 \times d q_{i}=0 \\
& -\sin \left(q_{3}+q_{7}\right) \times d q_{1}+\cos \left(q_{3}+q_{7}\right) \times d q_{2}+ \\
& +\left(L \times \cos q_{7}-b \times \sin q_{7}\right) \times d q_{3}+\underset{\stackrel{9}{\mathbf{a}}}{i=4} 0 \times d q_{i}=0 \\
& -\sin \left(q_{3}+q_{9}\right) \times d q_{1}+\cos \left(q_{3}+q_{9}\right) \times d q_{2}+ \\
& +\left(L \times \cos q_{9}-b \times \sin q_{9}\right) \times d q_{3}+\stackrel{\stackrel{9}{\mathrm{a}}}{i=4} 0 \times d q_{i}=0 \\
& \cos \left(q_{3}+q_{7}\right) \times d q_{1}+\sin \left(q_{3}+q_{7}\right) \times d q_{2}+ \\
& +\left(L \times \sin q_{7}-b \times \cos q_{7}\right) \times d q_{3}+\stackrel{\circ}{\mathbf{a}}_{i=4}^{5} 0 \times d q_{i}- \\
& -r_{r f} \times d q_{6}+\stackrel{9}{\stackrel{9}{\mathfrak{a}}} 0 \times d q_{i=7}=0 \\
& \cos \left(q_{3}+q_{9}\right) \times d q_{1}+\sin \left(q_{3}+q_{9}\right) \times d q_{2}+ \\
& +\left(L \times \sin q_{9}-b \times \cos q_{9}\right) \times d q_{3}+\underset{i=4}{\stackrel{5}{\boldsymbol{a}}} 0 \times d q_{i}- \\
& -r_{r f} \times d q_{8}+\stackrel{9}{\stackrel{\circ}{\boldsymbol{a}}} 0 \times d q_{i=7}=0
\end{aligned}
$$

Thus expression (9) represents the condition preventing translation along the $\mathrm{Y}_{\mathrm{R}}$-axis of the platform, equations (10) and (11) impose pure rolling without sliding of the driving wheels, expression (12) and (13) restrict sliding along the $y 3$-axes and $y 4$-axes of the driving wheels respectively, and expressions (14) and (15) also impose rolling without sliding of the turning wheels.

### 3.2 Direct kinematic model in Cartesian coordinates

For writing the kinematic equations of motion, each wheel is analyzed according to figure 5.

Assuming known angular velocities of the two driving wheels, in the study of the direct kinematic model, the unknowns are the linear
and angular velocity of the moving structure relative to the fixed reference system:

$$
\begin{align*}
& \dot{0}_{X}=\left[\begin{array}{lll}
\dot{x}_{p} & \dot{y}_{p} & \dot{\theta}
\end{array}\right]^{T}  \tag{16}\\
&=\left[\begin{array}{lll}
\dot{x}_{p} & \dot{y}_{p} & \omega
\end{array}\right]^{T}
\end{align*}
$$

Following some transformations, [6], the kinematic parameters of the points $O_{i}, i=1 \div 4$, representing the centers of the wheels of the mobile system $R$, projected onto the axes of the fixed reference system $O$ are expressed as :

$$
\begin{aligned}
& \bar{v}_{O_{1}}=\left[\begin{array}{c}
\dot{q}_{1}+l \cdot \dot{q}_{3} \cdot \cos q_{3} \\
\dot{q}_{2}+l \cdot \dot{q}_{3} \cdot \sin q_{3} \\
0
\end{array}\right] ; \omega_{1}=\left[\begin{array}{c}
0 \\
\dot{q}_{4} \\
\dot{q}_{3}
\end{array}\right] \text { ( } \\
& \dot{\bar{v}}_{01} \\
& =\left[\begin{array}{c}
\ddot{q}_{1}+l \cdot \ddot{q}_{3} \cdot \cos q_{3}-l \cdot \dot{q}_{3}{ }^{2} \cdot \sin q_{3} \\
\ddot{q}_{2}+l \cdot \ddot{q}_{3} \cdot \sin q_{3}-l \cdot \dot{q}_{3}{ }^{2} \cdot \cos q_{3} \\
0
\end{array}\right] ; \dot{\omega}(18 \\
& =\left[\begin{array}{c}
0 \\
\ddot{q}_{4} \\
\ddot{q}_{3}
\end{array}\right] \\
& \bar{v}_{O_{2}}=\left[\begin{array}{c}
\dot{q}_{1}-l \cdot \dot{q}_{3} \cdot \cos q_{3} \\
\dot{q}_{2}-l \cdot \dot{q}_{3} \cdot \sin q_{3} \\
0
\end{array}\right] ; \omega_{2}=\left[\begin{array}{c}
0 \\
\dot{q}_{5} \\
\dot{q}_{3}
\end{array}\right] \\
& \dot{\bar{v}}_{O_{2}} \\
& =\left[\begin{array}{c}
\ddot{q}_{1}-l \cdot \ddot{q}_{3} \cdot \cos q_{3}+l \cdot \dot{q}_{3}{ }^{2} \cdot \sin q_{3} \\
\ddot{q}_{2}-l \cdot \ddot{q}_{3} \cdot \sin q_{3}-l \cdot \dot{q}_{3}{ }^{2} \cdot \cos q_{3} \\
0
\end{array}\right] ; \dot{\omega}(20 \\
& =\left[\begin{array}{c}
0 \\
\ddot{q}_{5} \\
\ddot{q}_{3}
\end{array}\right] \\
& \bar{v}_{O_{3}} \\
& =\left[\begin{array}{c}
\dot{q}_{1}-L \cdot \dot{q}_{3} \cdot \sin q_{3}+b \cdot \dot{q}_{3} \cdot \cos q_{3} \\
\dot{q}_{2}+L \cdot \dot{q}_{3} \cdot \cos q_{3}+b \cdot \dot{q}_{3} \cdot \sin q_{3} \\
0
\end{array}\right] ; \omega(21 \\
& =\left[\begin{array}{c}
0 \\
\dot{q}_{6} \\
\dot{q}_{7}
\end{array}\right]
\end{aligned}
$$

$$
\begin{aligned}
& \dot{\bar{V}}_{O 3}=\left[\begin{array}{l}
\binom{\ddot{q}_{1}-L \cdot\left(\ddot{q}_{3} \cdot \sin q_{3}+\dot{q}_{3}{ }^{2} \cdot \cos q_{3}\right)+}{+b \cdot\left(\ddot{q}_{3} \cdot \cos q_{3}-\dot{q}_{3}{ }^{2} \cdot \sin q_{3}\right)} \\
\left(\ddot{q}_{2}+L \cdot\left(\ddot{q}_{3} \cdot \cos q_{3}-\dot{q}_{3}{ }^{2} \cdot \sin q_{3}\right)+\right. \\
+b \cdot\left(\ddot{q}_{3} \cdot \sin q_{3}+\dot{q}_{3}{ }^{2} \cdot \cos q_{3}\right) \\
0
\end{array}\right) ; \dot{\omega}_{3}=\left[\begin{array}{c}
0 \\
\ddot{q}_{6} \\
\ddot{q}_{7}
\end{array}\right]\left(\begin{array}{c}
22 \\
)
\end{array}\right. \\
& \bar{v}_{O_{4}} \\
& =\left[\begin{array}{c}
\dot{q}_{1}-L \cdot \dot{q}_{3} \cdot \sin q_{3}-b \cdot \dot{q}_{3} \cdot \cos q_{3} \\
\dot{q}_{2}+L \cdot \dot{q}_{3} \cdot \cos q_{3}-b \cdot \dot{q}_{3} \cdot \sin q_{3} \\
0
\end{array}\right] ; \omega(10 \\
& =\left[\begin{array}{c}
0 \\
\dot{q}_{8} \\
\dot{q}_{9}
\end{array}\right] \\
& \dot{\bar{V}}_{O 4}=\left[\begin{array}{l}
\binom{\ddot{q}_{1}-L \cdot\left(\ddot{q}_{3} \cdot \sin q_{3}+\dot{q}_{3}{ }^{2} \cdot \cos q_{3}\right)-}{-b \cdot\left(\ddot{q}_{3} \cdot \cos q_{3}-\dot{q}_{3}{ }^{2} \cdot \sin q_{3}\right)} \\
\left(\ddot{q}_{2}+L \cdot\left(\ddot{q}_{3} \cdot \cos q_{3}-\dot{q}_{3}{ }^{2} \cdot \sin q_{3}\right)-\right. \\
-b \cdot\left(\ddot{q}_{3} \cdot \sin q_{3}+\dot{q}_{3}{ }^{2} \cdot \cos q_{3}\right) \\
0
\end{array}\right) ; \dot{\omega}_{4}=\left[\begin{array}{c}
0 \\
\ddot{q}_{8} \\
\ddot{q}_{9}
\end{array}\right](23
\end{aligned}
$$

The previously obtained expressions characterize the direct kinematic model in Cartesian coordinates and which after applying matrix transformations will play an important role in determining the direct dynamics equations for the moving platform.

Based on the above considerations, [8], write the following relation in matrix form:

$$
\begin{align*}
\dot{\bar{X}}=\left[\begin{array}{c}
\dot{q}_{1} \\
\dot{q}_{2} \\
\omega=\dot{q}_{3}
\end{array}\right] & =\left[\begin{array}{cc}
c q_{3} & 0 \\
s q_{3} & 0 \\
0 & 1
\end{array}\right] \\
& \cdot\left[\begin{array}{c}
\frac{r}{2} \cdot\left(\dot{q}_{4}+\dot{q}_{5}\right) \\
\frac{r}{2 \cdot l} \cdot\left(\dot{q}_{4}-\dot{q}_{5}\right)
\end{array}\right] \tag{24}
\end{align*}
$$

which expresses the velocity of the robot relative to the fixed reference system $\{O\}$.

### 3.3 Determination of dynamic control functions for the moving structure

In order for the robot to execute the rectilinear translation motion, according to its constructive structure, see also figure 6 , the essential condition is that the driving moments of the driving wheels must be equal, i.e [9]:

$$
\begin{equation*}
Q_{m}^{4}=Q_{m}^{5} \tag{25}
\end{equation*}
$$

from which the following equivalences are deduced:

$$
\begin{align*}
& \left(\ddot{q}_{4}=\ddot{q}_{5}\right), q_{3}=\text { cst. },\left(\dot{q}_{3}, \ddot{q}_{3}\right) \\
& \quad=0,\left(\ddot{q}_{6}=\ddot{q}_{8}\right),\left(q_{7}, q_{9}\right)=0 \tag{26}
\end{align*}
$$

Following some transformations, [6], the expressions of the driving moments related to translation are:

$$
\begin{align*}
& Q_{m}^{4}=Q_{m}^{5}=\frac{r_{r s}}{2} \cdot\left[M \cdot\left(\ddot{q}_{1} \cdot \cos q_{3}+\ddot{q}_{2} \cdot \sin q_{3}\right)\right.  \tag{27}\\
& + \\
& \left.+\varepsilon \cdot\left(M_{r f} \cdot r_{r f}+M_{r s} \cdot r_{r s}\right)+\mu \cdot M \cdot g\right]
\end{align*}
$$

For the mobile platform to perform a rotation, we must write the expression (16):

$$
\begin{equation*}
\left(\dot{q}_{1}, \dot{q}_{2}\right)=0,\left(\ddot{q}_{1}, \ddot{q}_{2}\right)=0 \tag{28}
\end{equation*}
$$

Due to the mechanical constitution, according to the figure 6, another essential condition for achieving the orientation of the structure is:

$$
\begin{equation*}
Q_{m}^{4}=-Q_{m}^{5}, \tag{29}
\end{equation*}
$$

which leads to,

$$
\begin{equation*}
\left(\ddot{q}_{5}=-\ddot{q}_{4}\right),\left(\ddot{q}_{6}=\ddot{q}_{8}\right) \tag{30}
\end{equation*}
$$

Considering certain differential kinematic constraints of the mobile robot and following some transformations [6], it follows :

$$
\begin{equation*}
\ddot{q}_{6}=\frac{\sqrt{L^{2}+b^{2}}}{r_{r f}} \cdot \ddot{q}_{3} ; \ddot{q}_{4}=\frac{l}{r_{r s}} \cdot \ddot{q}_{3} \tag{31}
\end{equation*}
$$

So, according to [6], the expressions for the driving moments of the two driving wheels of the mobile robot in the case of orientation are as follows:

$$
\begin{align*}
& Q_{m}^{4}=-Q_{m}^{5}=\left[\frac{I_{\Delta_{P}}+2 \cdot M_{r f} \cdot\left(b^{2}+L^{2}\right)}{2 \cdot l}+\right. \\
& \left.+M_{r f} \cdot \frac{L^{2}+b^{2}}{2 \cdot l}+M_{r s} \cdot \frac{l}{2}\right] \cdot r_{r s} \cdot \ddot{q}_{3}+ \\
& +\frac{\mu \cdot M \cdot g \cdot r_{r s}}{2 \cdot\left[\mu \cdot\left(r_{r f}-r_{r s}\right)+L\right]} \cdot  \tag{32}\\
& \cdot\left[\left({ }^{R} x_{C}-\mu \cdot r_{r s}\right) \cdot \frac{\sqrt{L^{2}+b^{2}}}{l}\right. \\
& \left.\quad+\left(L-{ }^{R} x_{C}+\mu \cdot r_{r f}\right)\right]
\end{align*}
$$

Expression (32) is the one that represents the drive moments for the two driving wheels of the robotic system.

## 4. CONCLUSION

Starting from the idea of designing a structure with a role in the medical assistance of people with mobility impairments, a hybrid robot
structure was briefly presented, consisting of two mechanical structures: a mobile robot and a serial structure.

For the mobile structure, velocities and accelerations were determined, after which the expressions of the driving moments of the two driving wheels were determined. Their determination was performed considering the conditions necessary to achieve a translation or orientation of the robot, as well as kinematic constraints.

At the stage of dynamic control functions of any type, based on the movement of the structure along a well-determined path with the aim of reaching points, speeds and accelerations imposed by the work process, the motors of the drive system must overcome generalized external and gravitational forces, which are usually well determined, as well as generalized inertial forces due to motors, mechanical transmissions or the resistance structure of the mechanical system. The analysis of the displacement of the moving structure to reach the target point was based on the assumption that the positioning and orientation of the robot are not performed simultaneously, so the two motions were determined independently.

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# STABILIREA FUNCȚIILOR DE MIȘCARE PENTRU UN ROBOT ASISTENT DESTINAT PERSOANELOR CU DIZABILITĂȚI MOTORII 


#### Abstract

Rezumat: În zilele noastre, roboții și mecanismele robotizate reprezintă de asemenea o mare provocare tehnologică și ca urmare ele se regăsesc în toate domeniile de activitate, cum ar fi cele pentru servicii, management, industrie, agricultură și medicină. În domeniul medical, pe lângă roboții care îi ajută pe medici în diverse operații, există şi roboți care ajută pacienții cu diverse traumatisme și accidente vasculare cerebrale să îşi recupereze mobilitatea, astfel că implicarea și cererea de roboți în domeniul medical este în creştere. Robotul asistent prezentat în această lucrare are potențialul de a oferi sprijin pentru o gamă largă de sarcini legate de asistență fizică, terapii și reabilitare.


Claudiu SCHONSTEIN, Lecturer, Ph. D, Eng., Technical University of Cluj-Napoca, Department of Mechanical Systems Engineering, E-mail: Schonstein_claudiu@yahoo.com, No.103-105, Muncii Blvd., Office C103A, 400641, Cluj-Napoca.

Claudiu-Ioan RUSAN, Ph. D. Student Eng., Technical University of Cluj-Napoca, Department of Industrial Design Engineering and Robotics, email: claudiu.rusan@ipr.utcluj.ro, No.103-105, Muncii Blvd., phone: +40745/259492;

